

“Vibration Reduction Using Piezoceramics of Automotive Shafts”

¹Ms. Asha Kumawat
Asst.Prof. Mechanical Engg. Department
Poornima College of Engineering,Jaipur (RJ)

Dr.K.B.Waghulde
Professor, Mechanical Engg. Department
D.Y.Patil institute of Technology ,Pimpri, Pune.

ABSTRACT

Paper reports an experimental study on active vibration reduction for automotive shafts with the use of piezoelectric material. The work focuses on an axle of an Audi A2. The demand in the automobile sector for higher comfort in the vehicle is of a great importance alongside the requirements of lighter weight and low fuel consumption. These requirements are typically in conflict with each other. One solution is the use of intelligent materials instead of viscoelastic materials and proof mass absorbers. These solutions are quite heavy especially at low frequencies. Active vibration control and piezoelectric devices are advantageous in this application due to their low mass to performance ratio. Our research study explores the use of such piezoelectric devices for an axle. In conjunction with electronics it will reduce vibrations in the first natural bending mode of the axle. Laboratory tests simulated the condition present in the road. At first a stationary set up was used, then a simulated disturbance was input at the attachment points of the shaft. Finally, a test with rotating shaft was performed. Piezoelectric devices (custom Quick Packs from ACX, a Division of Cymer) were used as sensors and as actuators to properly control the axle during the different operating conditions. The power consumption of each actuator pair was less than 20W. The work described here details the test setup, the control strategy, the hardware implementation as well as the test results obtained.

Keywords- peizoceramics, automotive shaft, vibration, Viscoelastic materials etc.

Introduction

The demand in the automobile sector for higher comfort in the vehicle is of a great importance alongside the requirements of lighter weight and low fuel consumption. These requirements are typically in conflict with each other. Measurements presently employed to reduce the noise levels by using additional damping material are with a higher structural weight. Alternatives for the future are therefore being searched for. One solution is the use of intelligent materials instead of viscoelastic materials and proof mass absorbers. Viscoelastic materials quite heavy especially at low frequencies. Whereas intelligent materials persuade with their multifunctional properties. They are characterized by their ability to be used in shape with the structure as an actuator as well as a sensor. Besides other intelligent materials the piezoelectric ceramics seems to be the most promisingly ones for the use in automobile structures. Piezoelectric devices and active vibration reduction are advantageous in this application field due to their low mass to performance ratio. Our research study explores the use of such piezoelectric devices for an axle of an AUDI A2.



Figure 1: AUDI A2

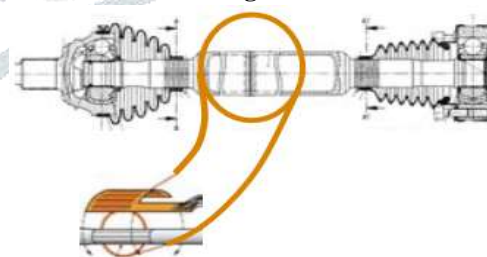


Figure 2: Applied piezoceramics on a test automotive shaft

Experimental Detail

The goal of this experimental study was the reduction of the first natural bending mode at 212Hz of the figured axle (Fig. 2) with the use of piezoceramic elements. At first a feasibility assessment of the piezo active control of the shaft was done by ACX Inc. Also a custom Quick Pack design

was developed that can be simply applied to the selected driveshaft.

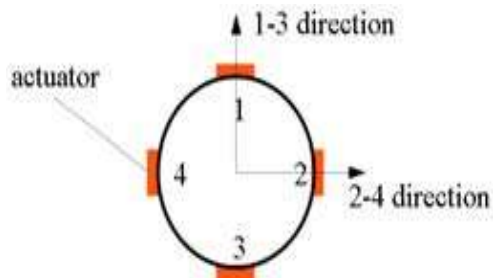


Fig. 3: Cross section of the shaft

The experimental setup consists of two independent channels of actuators. Each having two multilayer piezos diametrically opposed on the shaft and are placed nearly centered along the length of the shaft. The opposed actuators will be wired together, out of phase, such that a single voltage induces bending in that axis. It also contains 4 independent piezo strain sensors. It is positioned to the right of the actuator, yet close enough that physical collocation with the actuators is maintained. Figure 3 shows the cross section of the shaft and illustrates the relatively small amount of piezos required for active control of the shaft. The laboratory tests simulated the condition present in the road. Figure 4 shows the test setup used for the static (non-rotating) and dynamic (rotating) tests on the driveshaft. A simulated harmonic disturbance was input in the attachment points of the shaft. The ends were mounted as shown to simulate repeatable boundary conditions. The power consumption of each actuator pair was less than 20Watts. To transfer the information from the shaft and the required energy to the shaft, we used a slip ring solution.



Fig.4: stationary test set-up

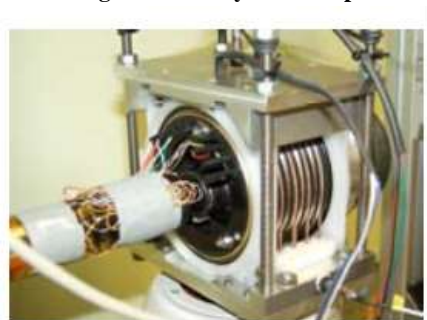


fig.5: slip ring solution

Shaft Parameters

- Shaft: Driveshaft AUDI A2
- Shaft Modulus: 210 Gpa
- Shaft Density: 7800 kg/m³
- Shaft Outer Radius: 19 mm
- Shaft Inner Radius: 15.5 mm
- Shaft Total Length: 680.2 mm
- Shaft weight: 5231 g
- Max. Deflection: 0.045 mm (at 212 Hz)
- Max. Rotations: 1440 RPM

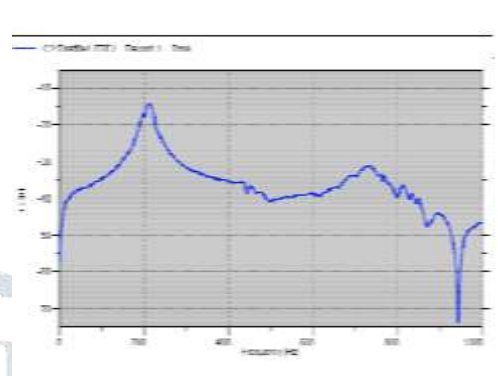


Fig. 6: open loop transfer function

Control

The investigation of the shaft showed that the influence of the actuators on the Signals of the 90 degree rotated Sensors is at a low level. Because of this we decided to establish 2 independent control loops, of or each of both actuators as shown in figure 7. Bcontrolloops are using the same type of control transfer function. Because of the equality of the plan of both control loops, the parameters of the two controller transfer functions are chosen to be equal.

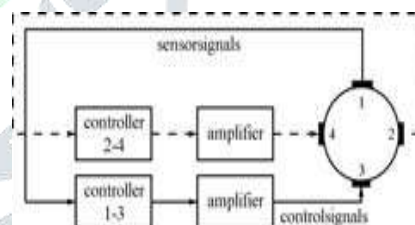


Fig. 7: Control approach for an active shaft

In order to get fast results an easy to tune collocated feedback controller proposed in was chosen to damp the first natural bending mode of the shaft. The structure of these controllers is simple and they can be implemented very easily using passive circuit elements and two op-amps. For testing a rapid control-prototyping-system has been used to implement the controller transfer functions digitally With a 10KHz sampling frequency. After tuning the controllers to the first bending mode several experiments were performed with non-rotating and rotating shaft. Figure 8 shows the steady State response of the rotating shaft to a 212 Hz excitation in open and closed loop operating mode. As can be seen a significant drop of the sensor voltage amplitude was achieved in closed loop operating mode.

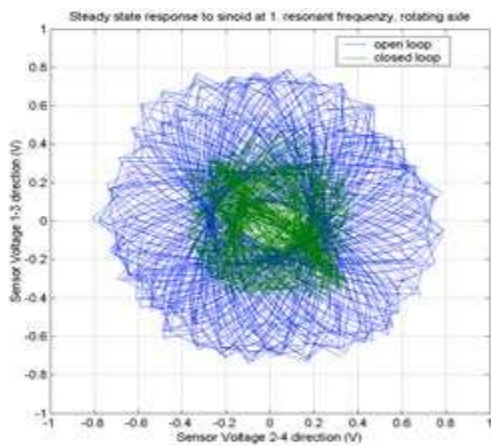


Figure 8: Experimental sinusoidal response for first mode (rotating shaft)

Further work

Further research work focuses on the miniaturization of the control- and power electronics. Furthermore a noncontact power transfer solution for rotating structures has to be found, studied and adapt to our problem. With piezoelectric devices applied to automotive shafts it should be possible to build lightweight shafts with excellent acoustic behavior.

Results

The work described here details the test setup, the control strategy as well as the test results obtained. Beyond these tests we performed the hardware implementation in the car. Finally we conducted experiments with the stationary car with rotating shaft as well as experiments driving the car on the road. The achieved results in reducing vibrations measured by accelerometers at the gearbox, the shock absorber and near the tire were impressive. As final test we performed measurements with microphones near the drivers and co-drivers ear. The obtained results show a reduction of interior sound pressure level of 12dB at the Eigen frequency of the shaft at 212 Hz.

References

- [1] Bailey T., Hubbard J.E.: Distributed Piezoelectric – Polymer Active Vibration Control of a Cantilever Beam. *Journal of Guidance, Control and Dynamics*, 8(5), 1985, pp. 605–611
- [4] Meng – Kao Yeh, Chih – Yuan Chin: Dynamic Response of Circular Shaft with Piezoelectric Sensor. *Journal of Intelligent Material Systems and Structures*, 5(11), 1994, pp. 833–840
- [6] Tylikowski A.: Dynamics of Laminated Beams with Active Fibers. 3rd Polish-German Workshop on Dynamical Problems in Mechanical Systems, 1993, pp. 67–78
- [7] Perini E. A., Bueno D. D., Santos R. B., Junior V. L., Nascimento Luiz de P. “Rotating Machinery Vibration Control Based on Poles Allocation using Magnetic Actuators” *International Conference on Engineering Optimization*, Rio de Janeiro, Brazil, June (2008), pp.1 -5
- [8] Bennett, Stuart “A history of control engineering”, (1993).

- [9] Ang K.H., Chong G.C.Y., and Li Y. “PID control system analysis, design, and technology”, *IEEE Trans Control Systems Tech*, vol.13, no.4, (2005), pp.559-576.
- [10] Foepl A., Der Civilingenieur, “Das Problem der Delaval’schen Turbinenvelle” (1895), pp. 333-342.
- [11] Stodola A., “Neve Kristische Drehzahlen als folye der kreisel wirkuag der laufrader” *Gesante Tubinenwes*, 15, (1918), PP. 269–275.
- [12] Jeffcott H. H., “The lateral vibration of loaded shafts in the neighbourhood of the whirling speeds”, *Philosophical magazine*, vol. 37, (1919), pp.304-315.
- [13] Kirk R. G., Gunte E. J., “the effect of support flexibility and damping on the synchronous response of a single mass flexible rotor”, *J. Eng. Ind., Trans. ASME* (1972) pp. 221–232.
- [14] Nelson, H.D. and McVaugh, J.M., “The Dynamics of Rotor-bearing Systems Using Finite Elements,” *ASME Journal of Engineering for Industry*, Vol. 98, No.2, (1976), pp. 593-600.
- [15] Yamamoto T., Yasuada K., and Nagasaka I. “Ultra-Subharmonic oscillations in a nonlinear vibratory system” *Bulletin of the JSME*, 19(138), (1976), pp.1442-1447.
- [16] Holmes A. G., Ettles C. M., and Mayes L. W., “Aperiodic behaviour of a rigid shaft in short journal bearings” *International Journal for Numerical methods in Engineering*, vol. 12, (1978), pp. 695-702, Panda K. C. and Dutt J. K., “Design of Optimum Support Parameters for Minimum