

II. LITERATURE REVIEW

The content of this paper is on overview of the available literature and information on hydraulic fixture that was used as foundation of this case study.

1. Barge K., et. al., [1] have presented design and development of hydraulic fixture for VMC. They have used hydraulic vertical swing clamps for holding the work piece driven by hydraulic power pack by replacing existing hydraulic fixture. The existing fixture replaced with hydraulic fixture to save time for loading and unloading of component
2. Raghu and Melkote., et. al., [2] have presented research paper on design, development and analysis of clamping force in fixture assembly in which workpiece location error is done by consideration of the fixture and workpiece due to fixturing forces. For modelling of workpiece, location error.
3. Sridharakeshava K. B. et. al., [3] has discussed about the general requirements of a fixture which includes constraints of deterministic location, contained deflection, geometry constraint in order to maintain the work piece stability during a machining process. They discussed 3 stages of fixture design, one deals with information gathering and analysis, second deals with product analysis and third involves design of fixture elements.

III. DESIGN OF FIXTURE

It is a systematic approach for the realization of total task. It comprises the theoretical analysis of the body, methods and principles associated with a branch of knowledge. Typically, it encompasses concepts such as paradigm, theoretical model, phases and quantitative or qualitative techniques.

A. Basic requirements

Study of part: The part study is most essential and initial step for the planer. The segment drawings are examined in order to find clamping zones of fixture parts.

Geometric Model: For this it is required to have all the basic measurement tolerance and result required.

Calculation: It includes calculation of different forces and direction going to act on Part while performing operation on it

Selection of tooling material: The material should be selected according to force required and also considering properties of part.

Cost estimation: Tool cost estimation is important particularly for little and medium clump of generation runs, where the expense of an instrument proclaims to a critical rate of the item cost advancing.

B. Component Details

The component is cylinder block of 4 stroke single cylinder SI engine made up of aluminum alloy AlSi 132, weighing 1.985 kg and component is produced by pressure die casting process. The operations to be performed on the component, using desinged fixture setup, are drilling and boring.

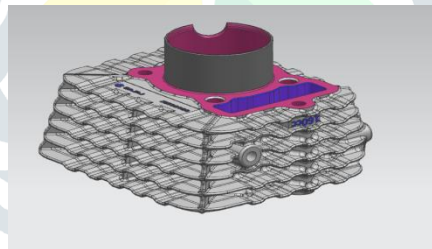


Fig 2. 3D CAD model of Cylinder Block

C. Computer Aided Fixture Designs

The clamping forces applied are caused to deform the workpiece, so therefore it is important to minimize such effects by using optimal design of the hydraulic fixture layout. In machining, work holding is a main concept and hydraulic fixture is the element responsible to satisfy this. Locating, orienting, centering, supporting and clamping can be considered as basic function requirements of fixtures.

For this engine cylinder block hydraulic fixture is designed in NX 11 software by considering all locating and clamping principles. The fixture is having two locating pins, three resting pads and two vertical hydraulic clamps.

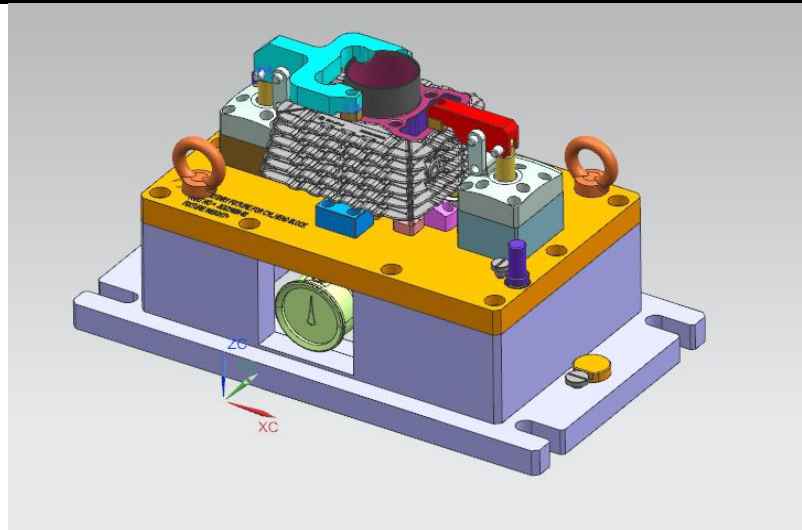


Fig 3. 3D CAD model of Fixture assembly with cylinder block

D. Clamping forces calculations by analytical method

The clamping forces by numerical and analytical method are to be calculated at given hydraulic pressures of 40, 50 and 60 bar.

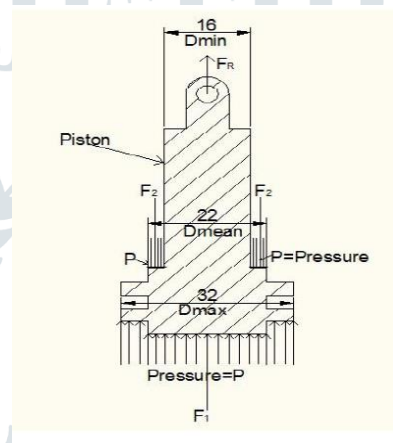


Fig 4. Boundary conditions applied on piston

By considering free body diagram as shown in fig with clamping force and hydraulic pressure

Hydraulic pressure= 4MPa

D_{min} = 32mm

D_{min} =16mm

D_{mean} =22mm

Force acting on piston by hydraulic pressure

$$F_1 = \pi/4 \times D_{max}^2 \times P$$

$$= 3.126 \text{ KN}$$

Force acting on piston head by cylinder head

$$F_2 = \pi/4 \times (D_{mean}^2 - D_{min}^2) \times P$$

$$= \pi/4 \times (22^2 - 16^2) \times 4$$

$$= 0.715 \text{ KN}$$

Net force acting

$$= F_1 - F_2$$

$$= 3.126 - 0.715$$

$$= 2.5 \text{ KN}$$

By considering clamp lever a rigid element with negligible mass forces acting on beam will be

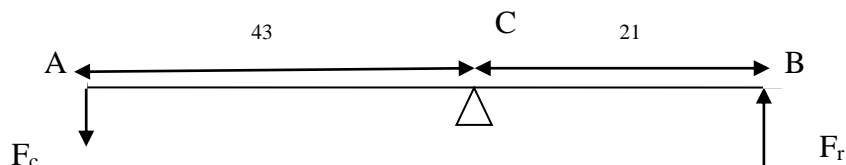


Fig 5. Free body diagram of single clamp lever

C= point where lever is hinged

F_H= reaction force at hinge

B= point where net piston force is applied F_R

A= point where clamping force is applied

From free body diagram taking moment about C

$$\sum M_C = 0$$

$$F_c \times 43 = F_r \times 21$$

$$F_c = 2.5 \times 21/43$$

$$F_c = 1.22 \text{ KN}$$

Total reaction force

$$F_H = F_C + F_R$$

$$= 3.72 \text{ KN}$$

Similarly clamping forces for 5 and 6 MPa hydraulic pressure are calculated and those are 1.526 KN and 1.831 KN respectively

For clamp 2

C = point where lever is hinged

F_H= reaction force at hinge

B= point where net piston force is applied F_R

A= point where clamping force is applied

F_{C1} and F_{C2} are individual clamping force given by both sides of C- clamp

For calculation it is required to transfer both F_{C1} and F_{C2} along the center line of the clamp

In order to transfer these forces, transferring angular moments of these forces at center line is required, angular moments are equal and opposite because of symmetry so they are cancelled. Now C-clamp is converted into simple beam clamp with clamping force F_C = F_{C1} + F_{C2}.

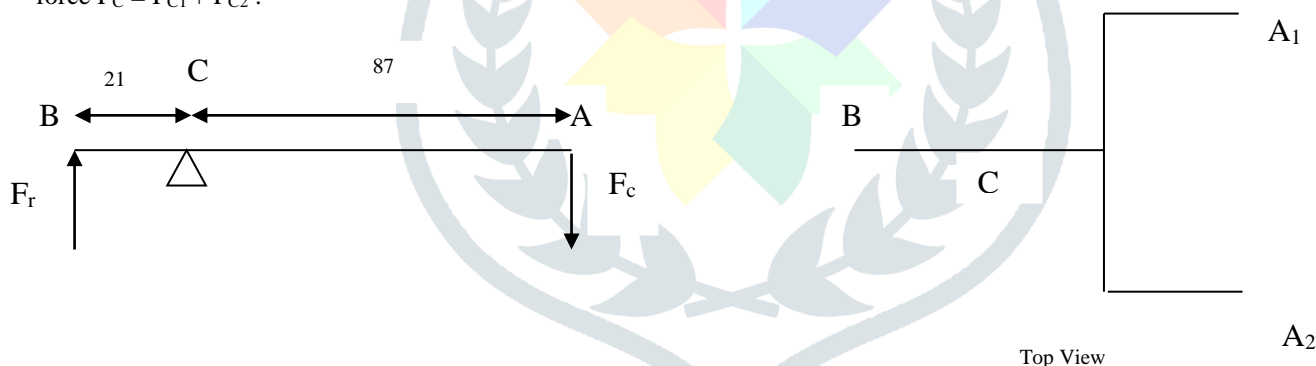


Fig 6. Free body diagram of C-clamp lever

From free body diagram

Taking moment about C

$$\sum M_C = 0$$

$$F_c \times 87 = F_R \times 21$$

$$F_c = 2.5 \times 21/87$$

$$F_c = 0.6037 \text{ KN}$$

$$F_{C1} = F_{C2} = F_C / 2 = 0.301 \text{ KN}$$

Similarly clamping forces for 5 and 6 MPa hydraulic pressure are calculated and those are 0.377 KN and 0.452 KN respectively.

E. Development and Experimentation of manufactured fixture

E.1 Development

Here in final CAD model of complete fixture assembly the two single clamps are converted into C-clamp and hence by using one C-clamp and one single hydraulic vertical clamp, total required hydraulic cylinders are reduced to two and thus cost of entire fixture assembly is reduced.

After manufacturing of all parts used in fixture and by assembling it total weight of fixture is 25.74 kg. It is manufactured according to given requirement within given tolerance limits.



Fig. 7 Hydraulic fixture Assembly

E.2 Experimentation of Manufactured Fixture

For testing deformations in cylinder block at different clamping forces for different hydraulic pressures used a dial gauge indicator for deflection measurement is used.

Least count of dial indicator is 0.01mm. For upward movement of lever ball pointer on dial scale deflects towards clockwise direction and for downward movement of lever ball pointer deflects on anticlockwise direction.



Fig. 8 Dial gauge indicator mounted on fixture

F. Results and Discussions

From experimental testing by using dial indicator following results of total deformation at different clamping loads for hydraulic pressures of 40, 50 and 60 bars are found

Table I- Deformation table

Hydraulic Pressure (bar)	Clamping Force (KN)	Deformation (Micron)
40	0.301	15.18
50	0.377	21.89
60	0.452	26.54

Table II- Benefits of using hydraulic fixture

Sr. No	Parameters	Manual fixture	Hydraulic Fixture
1	Loading/Unloading time	1min 25 sec	25 sec
2	Cycle time/job	5 min	2 min
3	Jobs/shift	70	90
4	Rejection % of 1 product	3%	1%

Major benefits of this research work

As three hydraulic clamps get reduce to two clamps that are one C-clamp and one single vertical clamp, cost of one hydraulic cylinder is reduced saving 11000/- in total manufacturing of fixture.

IV. CONCLUSIONS

In this paper design requirements of hydraulic fixture were studied and accordingly fixture was design in NX11 software and deformation of component is calculated by Dial gauge indicator. Clamping forces are calculated at required optimum hydraulic pressure using numerical and analytical methods.

At 40 bar hydraulic pressure cylinder block is perfectly clamped and maintained its position as well as stability against cutting forces with minimum deformation of 15.18 micron. Hence for clamping 40 bar hydraulic pressure is selected over 50 and 60 bar. Thus suggested fixture helps in achieving precise, reliable and accurate production methods.

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