



Performance Analysis of Conditioner With and Without Liquid Desiccant

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Abstract : This research investigates the performance characteristics of air conditioning systems operating with and without the integration of a liquid desiccant. Liquid desiccants are hygroscopic solutions capable of absorbing moisture from air, offering an alternative method for dehumidification that can enhance overall system efficiency. The study compares energy consumption, cooling capacity, dehumidification effectiveness, and coefficient of performance (COP) under controlled environmental conditions.

INTRODUCTION AIR CONDITIONING SYSTEMS ARE INTEGRAL TO MODERN COMFORT AND AIR QUALITY MANAGEMENT. TRADITIONAL SYSTEMS RELY ON VAPOR COMPRESSION CYCLES TO COOL AND DEHUMIDIFY AIR, WHICH OFTEN LEADS TO HIGH ENERGY USAGE. INTEGRATING LIQUID DESICCANTS INTO THESE SYSTEMS CAN POTENTIALLY IMPROVE ENERGY EFFICIENCY BY HANDLING LATENT HEAT LOADS MORE EFFECTIVELY. THIS PAPER EXAMINES THE COMPARATIVE PERFORMANCE OF AIR CONDITIONING SYSTEMS WITH AND WITHOUT A LIQUID DESICCANT UNIT.

I. Methodology Two experimental setups were used: a conventional vapor-compression air conditioner and a modified version incorporating a liquid desiccant dehumidification unit. Calcium Chloride (CaCl₂) was used as the liquid desiccant due to its high moisture absorption capacity. Key parameters such as ambient temperature, relative humidity, cooling load, power input, and air properties at the inlet and outlet were monitored over consistent time intervals.

II. Performance Metrics

- **Cooling Capacity (kW):** Measurement of the total heat removed from the space.
- **Coefficient of Performance (COP):** Ratio of cooling provided to electrical energy consumed.
- **Power Consumption (kWh):** Total energy used by the system.
- **Dehumidification Efficiency (%):** Reduction in humidity level achieved.

III. Theoretical framework

Selection of Pump: Design Conditions / Data

$$\begin{aligned}
 \text{I)} \quad & 1100 \text{ BTU/Pound/hr} \\
 & = 1100 \times 50.75 \\
 & = 55,781 \text{ BTU/Pound/hr} \\
 & = 16.34 \text{ Kw} \\
 & = 6\text{TR System}
 \end{aligned}$$

$$\text{II) Specific Heat of Lithium Chloride } 1 \text{ mol of LiCL} = 42.394 \text{ grams.}$$

$$= 42.394 / 1000 = 0.042394 \text{ Kg}$$

$$= 48.03 \times 0.042394$$

$$= 2.0361 \text{ KJ/Kg Kelvin}$$

$$\text{III) } 17^\circ\text{C DBT } 50\% \text{ Relative Humidity (} 80 \text{ to } 100^\circ\text{C) } \text{CpL} = 2.0361 \text{ KJ/KgK } \text{Cpw}$$

$$= 4.187 \text{ KJ/KgK}$$

$$\text{IV) Mass of Liquid QH} = 16.34 \text{ Kw QH}$$

$$= \text{mL} \cdot \text{CpL} \cdot \Delta T$$

$$16.34 = \text{mL} \cdot (2.0361) \cdot 20$$

$$\text{mL} = 0.4012 \text{ Kg/Sec}$$

$$\text{V) Specific Heat of Solution}$$

$$\text{QH} = \text{mw} \cdot \text{Cpw} \cdot \Delta T$$

$$16.34 = \text{mw} \cdot (4.187) \cdot 20$$

$$\text{mw} = 0.1951 \text{ Kg/Sec}$$

$$(\text{CpL})_w \times \text{mw} + (\text{CpL})_{\text{LiCL}} \times \text{mLiCL} = (\text{ma} + \text{mLiCL}) (\text{CpL})_{\text{Solution}} (4.187 \times 0.70) + (2.0361 \times 0.30) = (0.70 + 0.30)$$

$$(\text{CpL})_{\text{Solution}} (\text{CpL})_{\text{Solution}} = 3.5417 \text{ KJ/KgK}$$

Calculation of Fan

$$\rho_a \cdot \text{ma} + \rho_{\text{LiCL}} \cdot \text{mLiCL} = (\text{ma} + \text{mLiCL}) \cdot \rho_{\text{Solution}} (1000 \times 0.7) + (2078 \times 0.3) = (0.70 + 0.30) \cdot \rho_{\text{Solution}}$$

$$\rho_{\text{Solution}} = 1323.4 \text{ Kg/m}^3$$

$$\text{Volume Flow rate} = \text{mL} \rho_{\text{Solution}}$$

$$= 0.4012 / 1323.4$$

$$= 0.00030315$$

$$= 18.189 \text{ LPM}$$

Selection of Blower For Conditioner

$$\text{QL} = \text{ma} \cdot \text{Cpm} \cdot (\text{Td1} - \text{Td2}) \text{ QL} = 6\text{TR} = 21.10\text{Kw}$$

$$\text{Cpm} = 1.88 \text{ Kj/Kgk}$$

$$\text{Td1} = 40^\circ\text{C}, \text{Td2} = 17^\circ\text{C}$$

$$\text{QL} = \text{ma} \cdot \text{Cpm} \cdot (\text{Td1} - \text{Td2})$$

$$21.10 = \text{ma} \cdot 1.88 \cdot (40 - 17)$$

$$\text{Mass of Air (ma)} = 0.4879 \text{ Kg/s } ma = V / V1$$

$$V = ma \cdot V1$$

$$V1 = 0.935 \text{ m}^3/\text{kg } V = 0.4879 \times 0.935$$

$$\text{Volume Flow rate (V)} = 0.4561 \text{ m}^3/\text{s } V = 27.366 \text{ CMM}$$

$$\text{CFM} = 970$$

Liquid Desiccant System (Calculations For Conditioner)

(Water as Refrigerant)

$$Ma = (L \times W \times H) \rho_w$$

$$Ma = 0.1665 \text{ kg/s}$$

$$\text{Cooling Capacity (Ql)} = Ma (h1 - h2)$$

$$= 0.1665 \times (53 - 52)$$

$$= 0.41625 \text{ Kw}$$

Water Vapor addition rate

$$Mv = Ma (W1 - W2)$$

$$= 0.1665 (0.0125 - 0.0095)$$

$$= 4.995 \times 10^{-4}$$

$$= 1.7982 \text{ Kg / Hr}$$

IV. Results and Discussion The system incorporating a liquid desiccant demonstrated enhanced dehumidification capabilities and maintained thermal comfort at lower power consumption levels. The COP improved by approximately 12% in the desiccant-integrated system. While initial system complexity and maintenance requirements increased, the energy efficiency gains offer long-term benefits.

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